

# THE DRAG-REDUCTION OSCILLATING-WALL PROBLEM: NEW INSIGHT AFTER 20 YEARS

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## ACTIVE OPEN-LOOP TECHNIQUE

Energy input into system

Pre-determined forcing

Channel flow DNS ( $Re_\tau = u_\tau h/\nu = 200$ )

## SPANWISE WALL OSCILLATIONS

New approach: *Turbulent enstrophy*

*Transient evolution*

## CONSTANT DP/DX

$\tau_w$  is fixed in fully-developed conditions

**GAIN:**  $U_b$  increases

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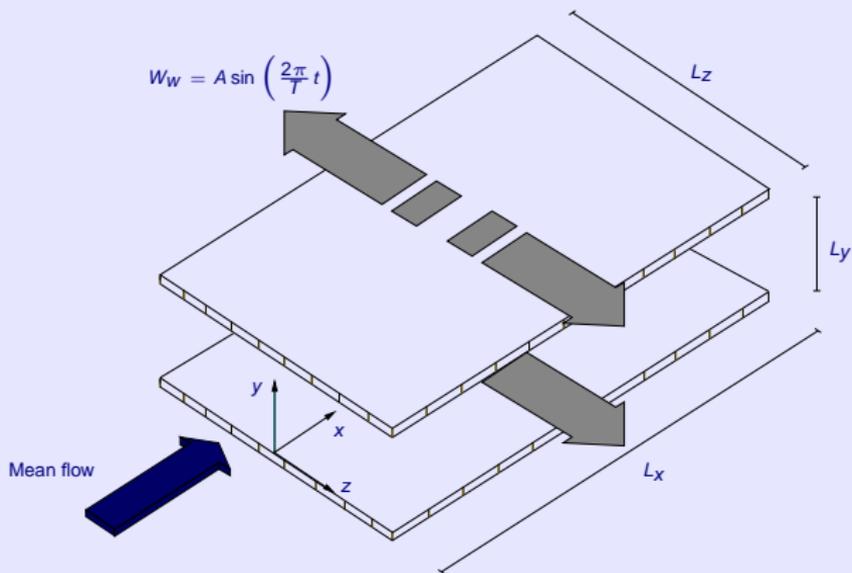
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## GEOMETRY

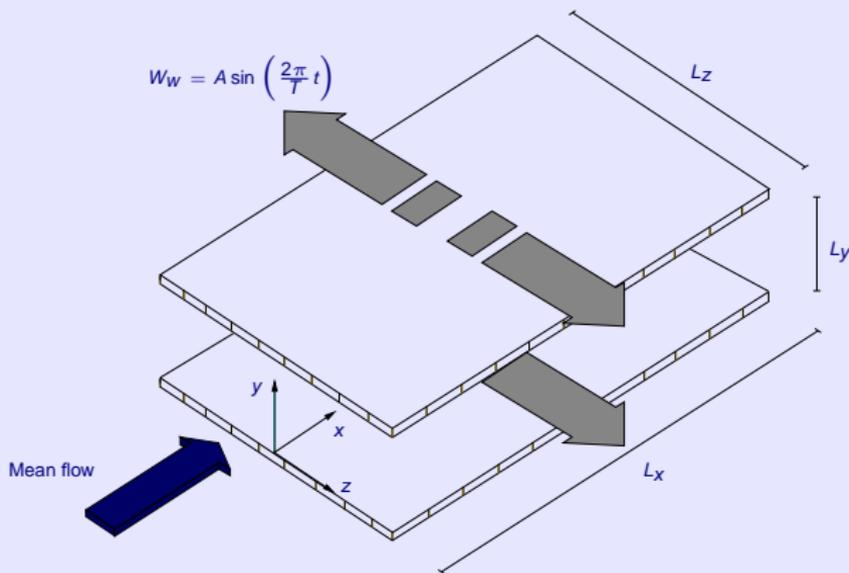


$$R = \frac{C_{f,r} - C_{f,o}}{C_{f,r}} = \frac{U_{b,o}^2 - U_{b,r}^2}{U_{b,o}^2}$$

Why does the skin-friction coefficient decrease?

$C_f = \tau_w / (1/2 \rho U_b^2)$  decreases  $\rightarrow$  study why  $U_b$  increases

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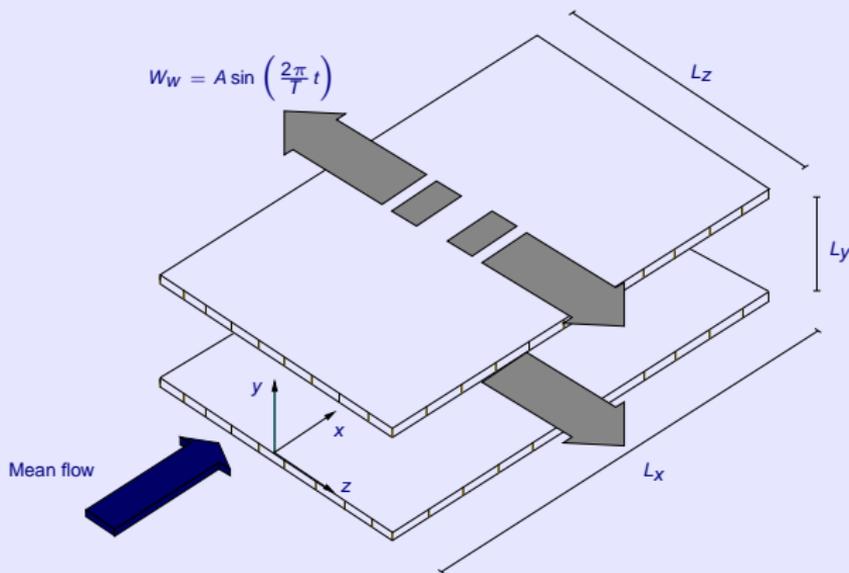


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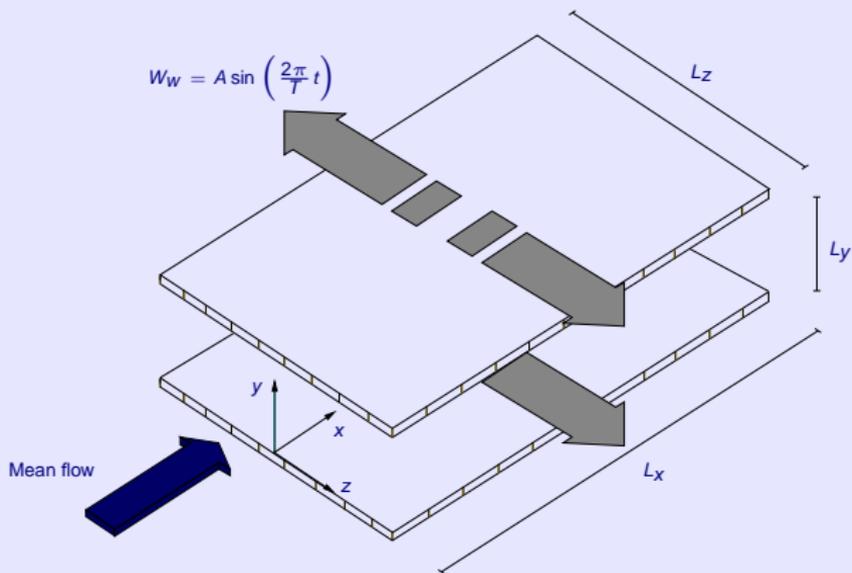
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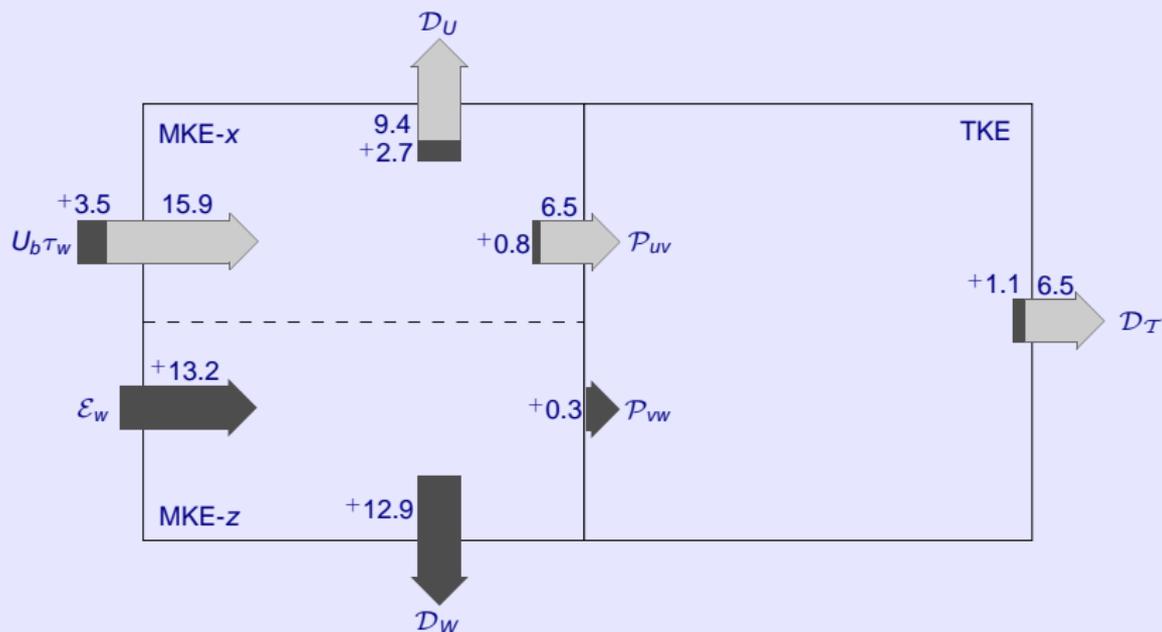


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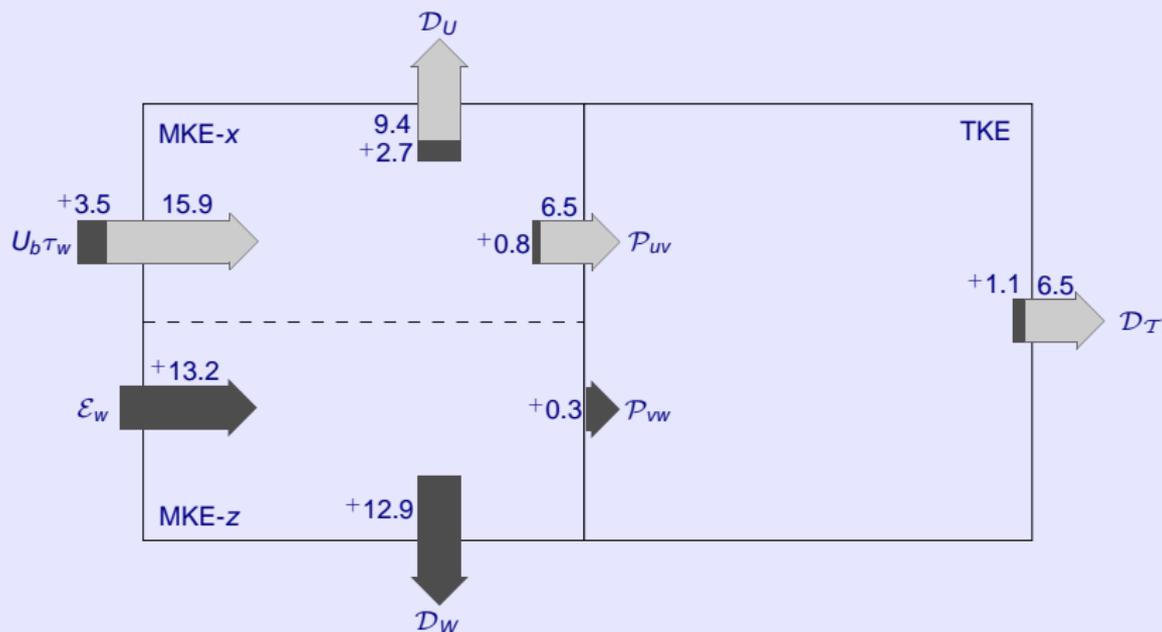
Energy is fed through  $P_x$  ( $\rightarrow U_b \tau_w$ ) and wall motion ( $\rightarrow \mathcal{E}_w$ )

Energy is dissipated through:

Mean-flow viscous effects ( $\rightarrow \mathcal{D}_U, \mathcal{D}_W$ )

Turbulent viscous effects ( $\rightarrow \mathcal{D}_T$ )

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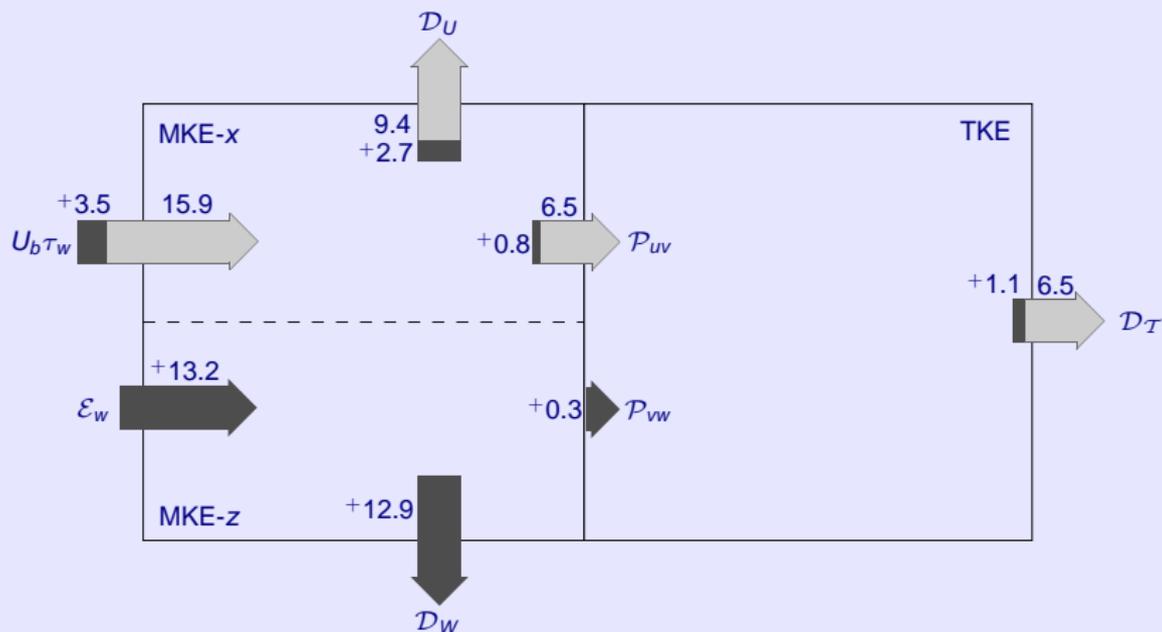


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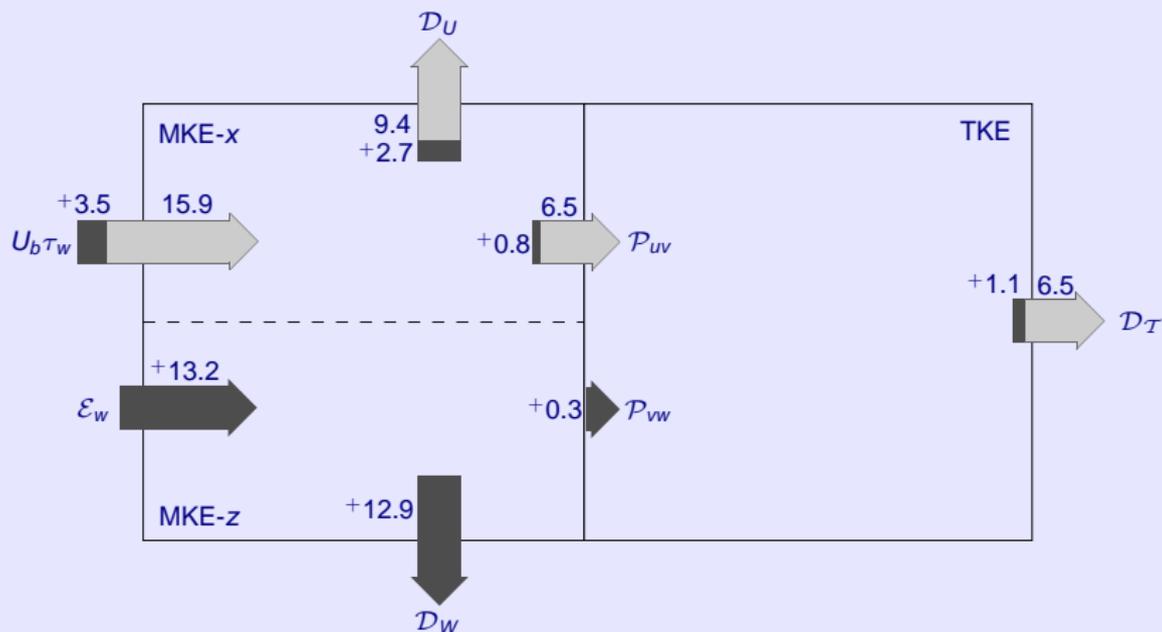


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## STILL TO BE ANSWERED

Why does TKE decrease?

Why does  $U_b$  increase?

DOES  $W$  ACT ON TURBULENT DISSIPATION?

- Stokes-layer-type flow is generated by the wall oscillation
- Stokes layer's direct action on  $\mathcal{D}_T = \int_V \widehat{\omega_i \omega_i} dV$
- Study the transport of turbulent enstrophy  $\widehat{\omega_i \omega_i}$
- The term *enstrophy* was coined by G. Nickel and is from Greek  $\sigma\tau\rho\phi\acute{\eta}$   
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## Terms scaled in viscous units

Stokes layer influences dynamics of turbulent enstrophy

Three terms: which is the dominating one?

→ Let's look at the terms of the equation

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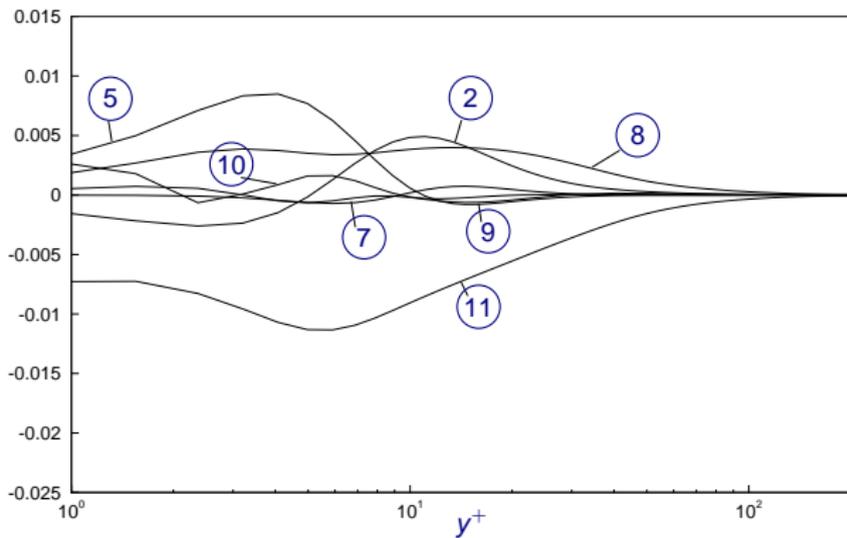
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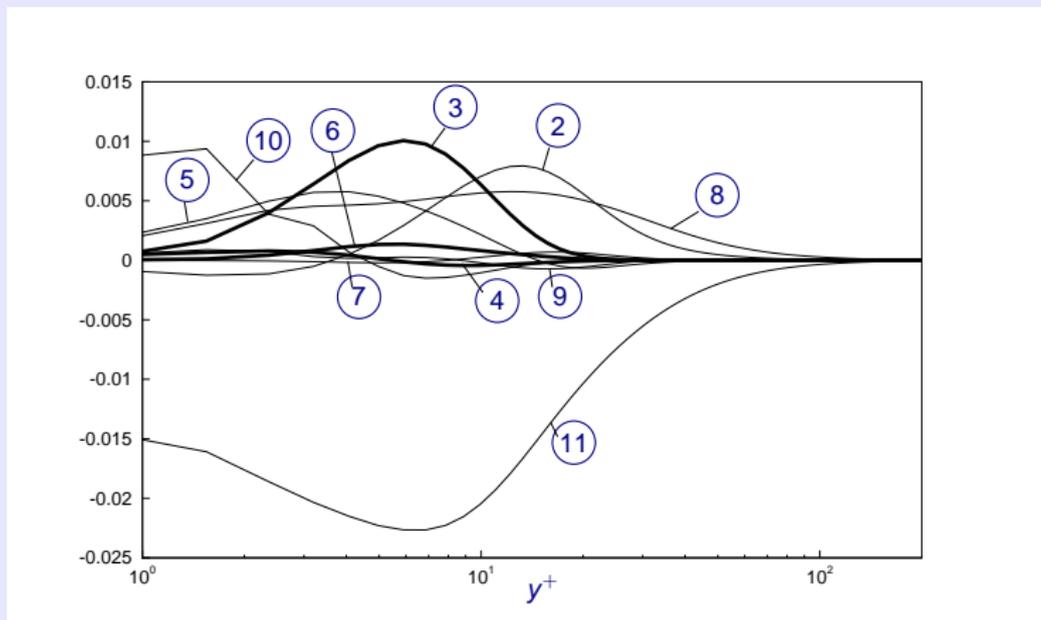
# TURBULENT ENSTROPY PROFILES

FIXED WALL



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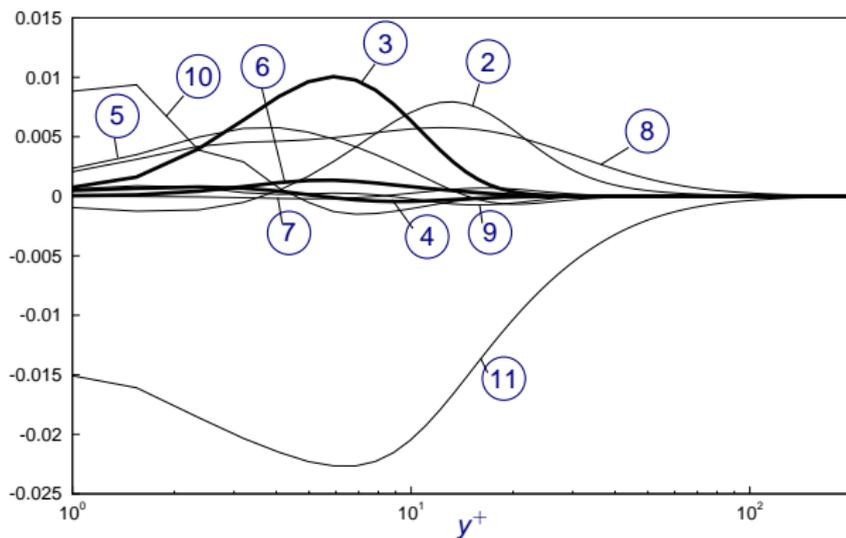
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Term 3,  $\widehat{\omega_z \omega_y} \partial \widehat{W} / \partial y \rightarrow$  turbulent enstrophy production is dominant  
Turbulent dissipation of turbulent enstrophy increases

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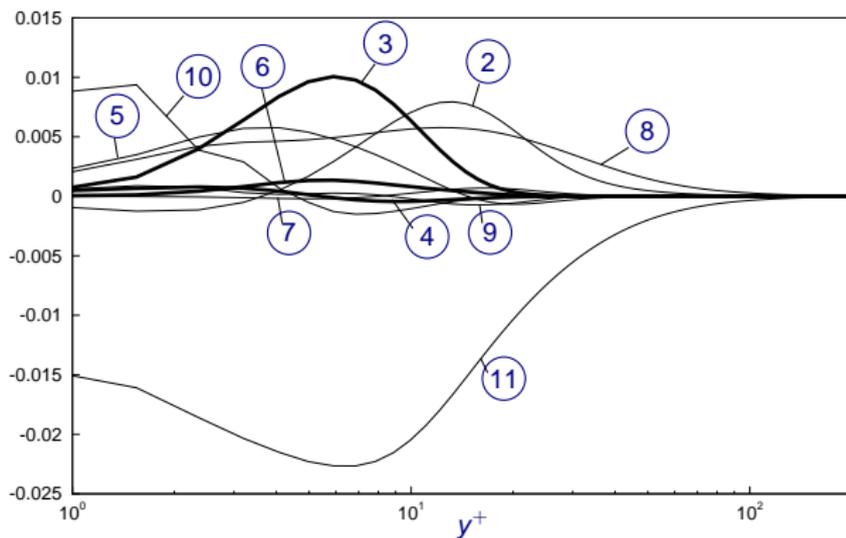


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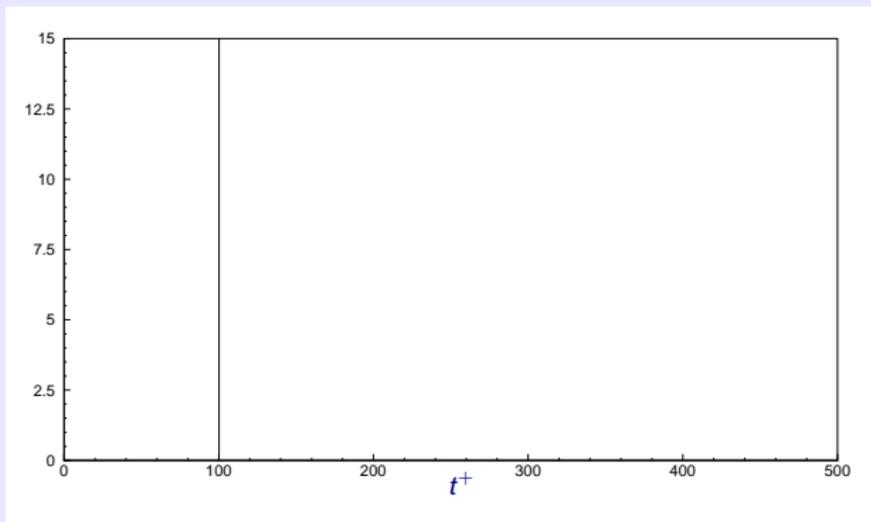
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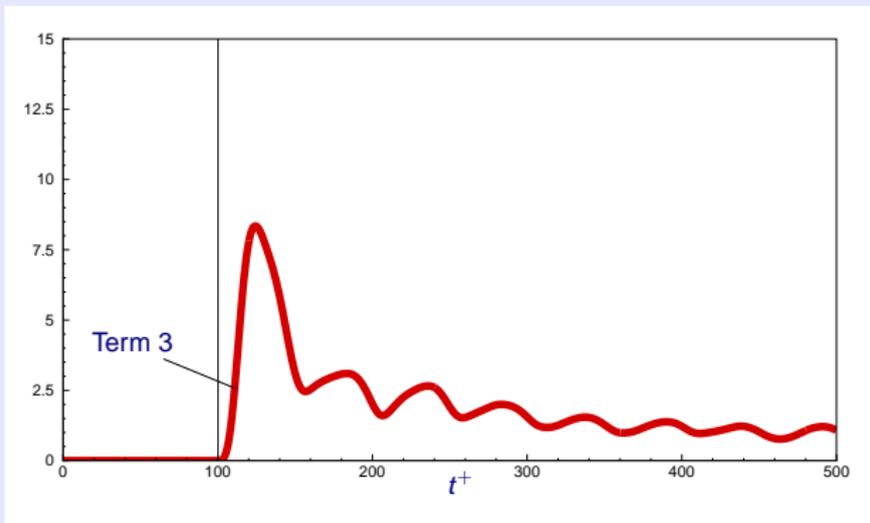
Key: transient from start-up of wall motion



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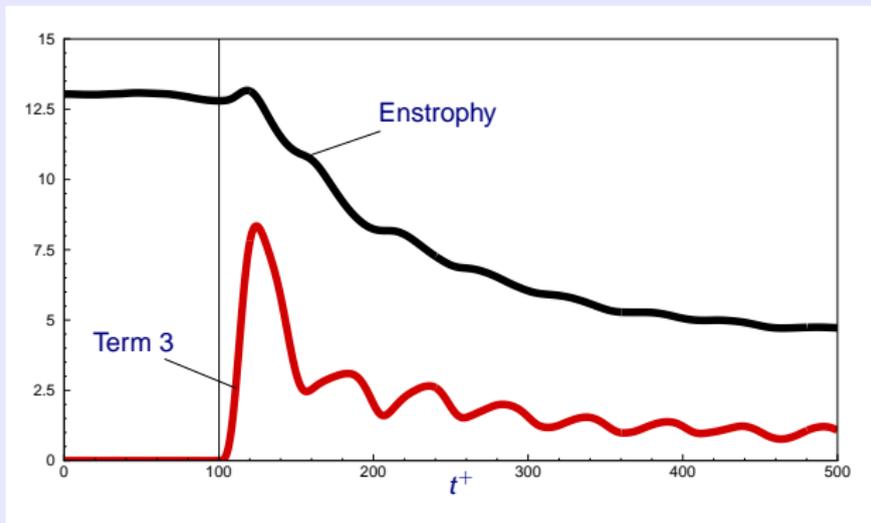


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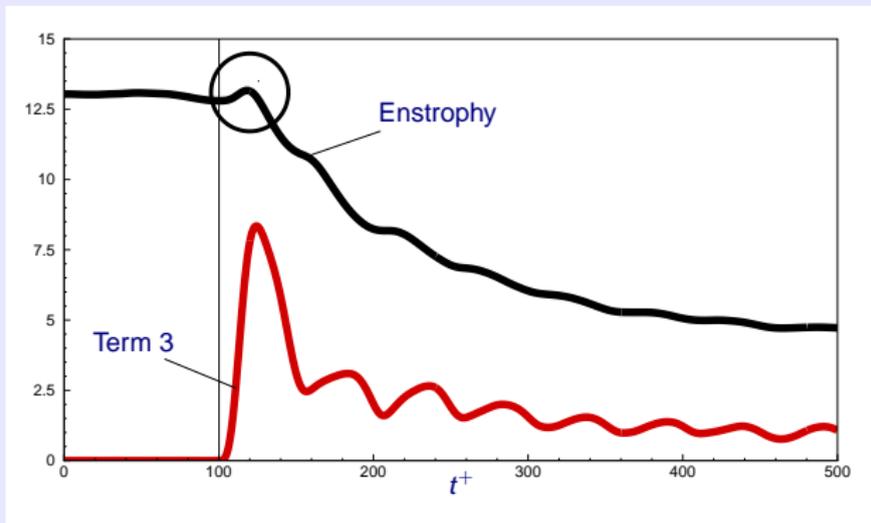
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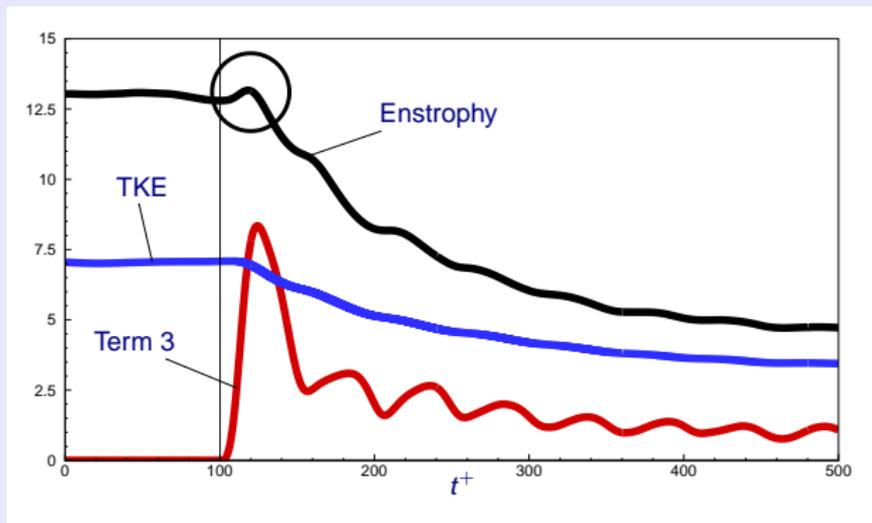
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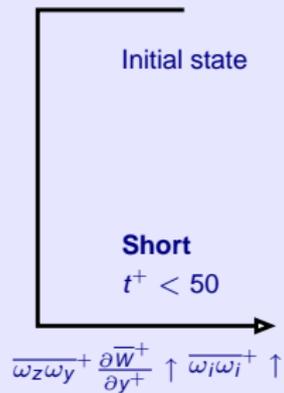
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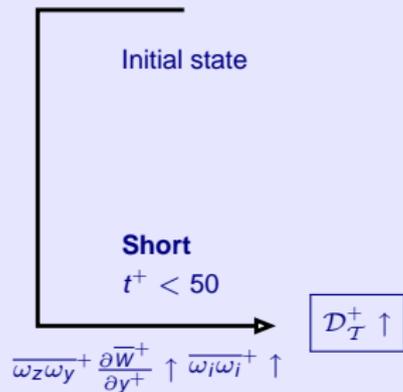
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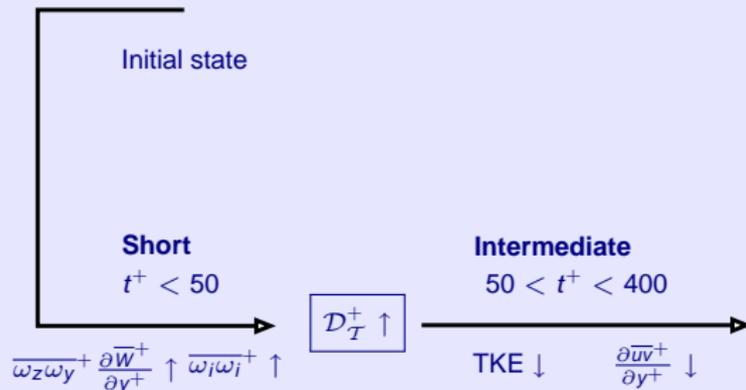
**BLACK:** turbulent enstrophy increases, then decreases

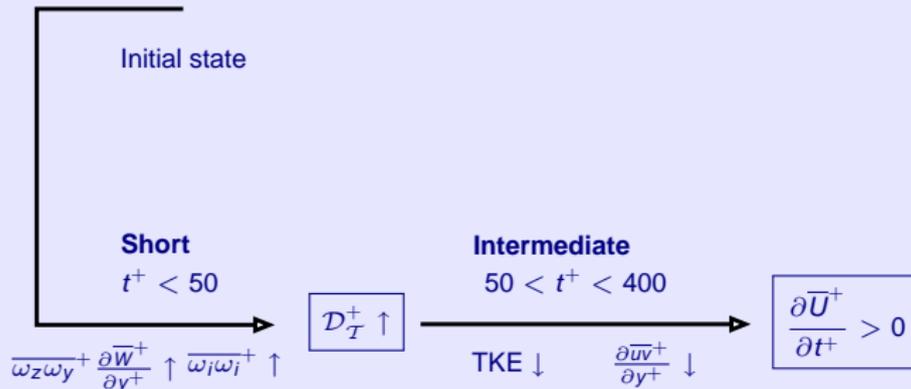
**BLUE:** TKE decreases monotonically

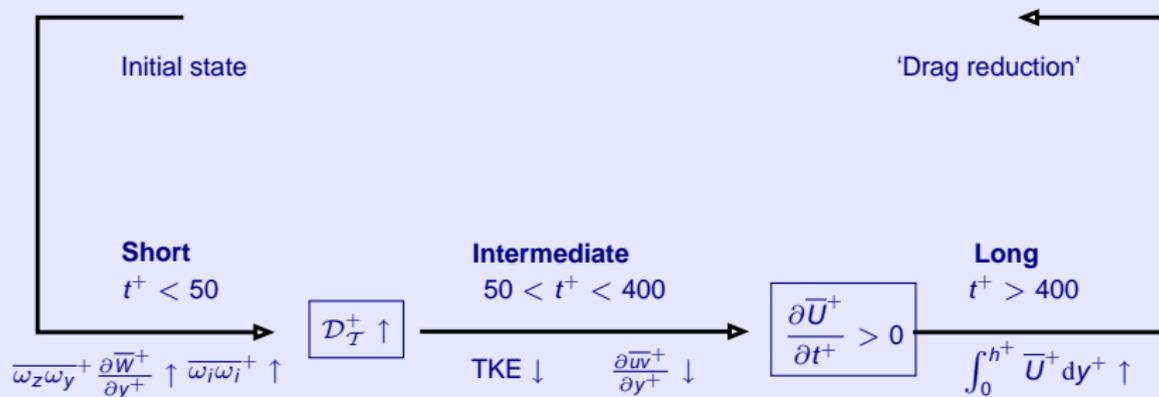
Initial state

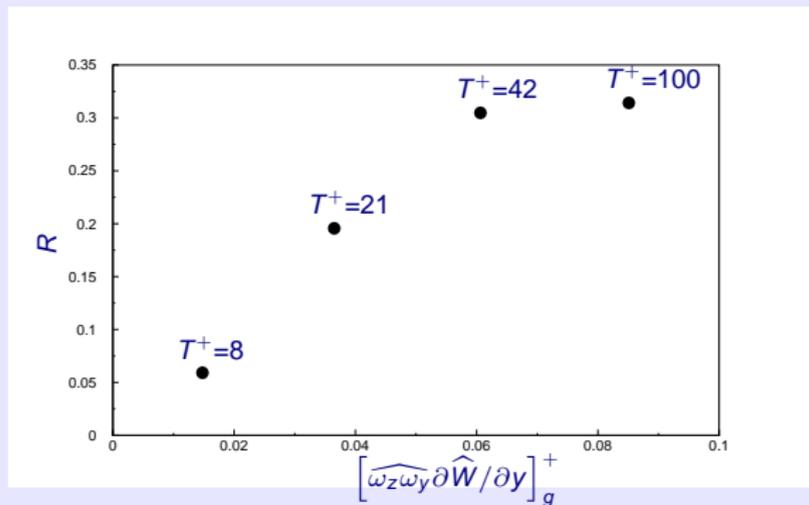












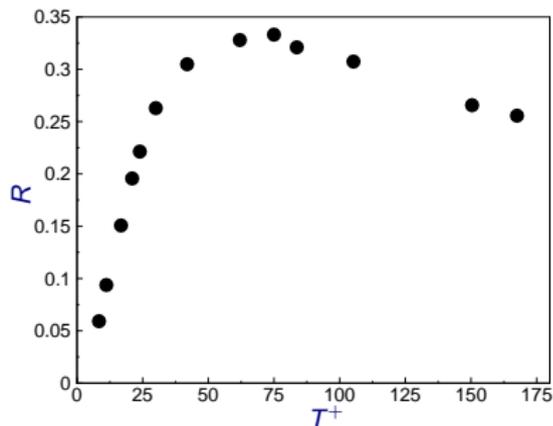
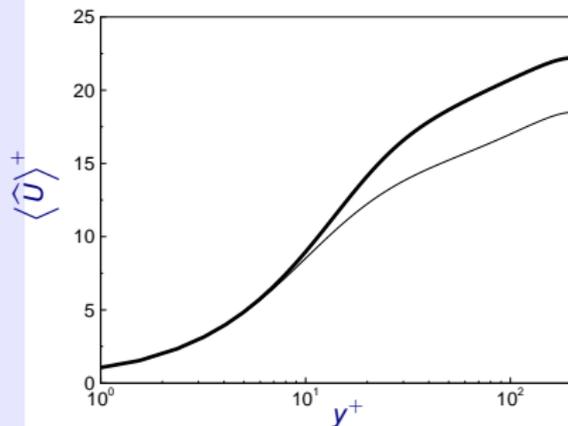
Drag reduction grows monotonically with global production term

This happens up to optimum period

THANK YOU!

#### REFERENCE

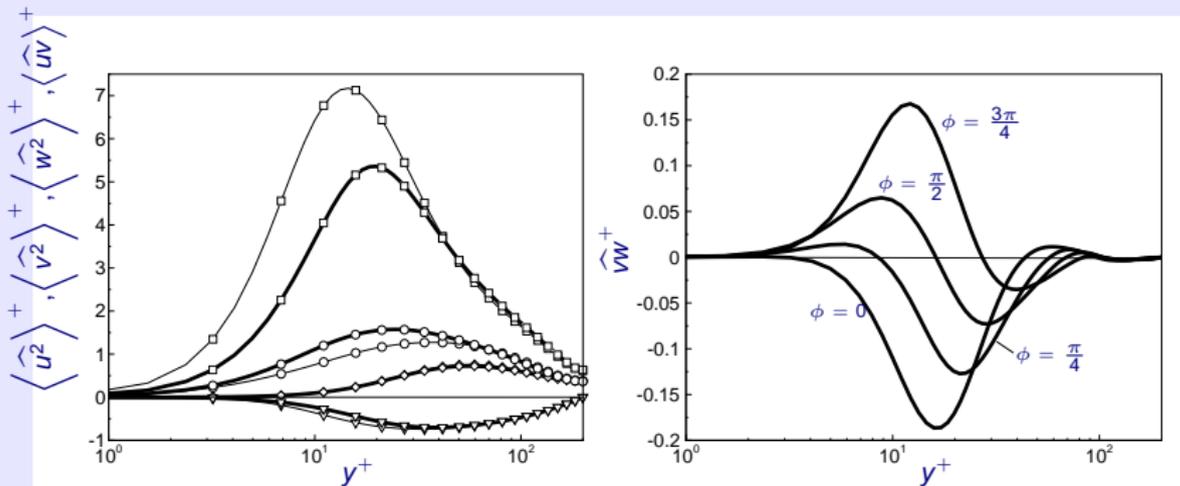
Ricco, P. Ottonelli, C. Hasegawa, Y. Quadrio, M.  
Changes in turbulent dissipation in a channel flow with oscillating walls  
*J. Fluid Mech.*, 700, 77-104, 2012.



Mean velocity increases in the bulk of the channel

Mean wall-shear stress is unchanged

Optimum period of oscillation  $T^+ \approx 75$



Turbulence kinetic energy decreases

Streamwise velocity fluctuations are attenuated the most

New oscillatory Reynolds stress term  $\widehat{vw}$  is created,  $\langle \widehat{vw} \rangle = 0$

## GLOBAL MEAN KINETIC ENERGY EQUATION

$$U_b^+ \tau_w^+ + \underbrace{\left\langle A^+ \frac{\partial \widehat{W}^+}{\partial y^+} \Big|_{y^+=0} \right\rangle}_{\varepsilon_w} = - \underbrace{\left[ \widehat{uv}^+ \frac{\partial \widehat{U}^+}{\partial y^+} \right]_g}_{\mathcal{P}_{uv}} - \underbrace{\left[ \widehat{vw}^+ \frac{\partial \widehat{W}^+}{\partial y^+} \right]_g}_{\mathcal{P}_{vw}} + \underbrace{\left[ \left( \frac{\partial \widehat{U}^+}{\partial y^+} \right)^2 \right]_g}_{\mathcal{D}_U} + \underbrace{\left[ \left( \frac{\partial \widehat{W}^+}{\partial y^+} \right)^2 \right]_g}_{\mathcal{D}_W}$$

## GLOBAL TURBULENT KINETIC ENERGY EQUATION

$$\underbrace{\left[ \widehat{uv}^+ \frac{\partial \widehat{U}^+}{\partial y^+} \right]_g}_{\mathcal{P}_{uv}} + \underbrace{\left[ \widehat{vw}^+ \frac{\partial \widehat{W}^+}{\partial y^+} \right]_g}_{\mathcal{P}_{vw}} + \left[ \frac{\partial \widehat{u}_i^+}{\partial x_j^+} \frac{\partial \widehat{u}_i^+}{\partial x_j^+} \right]_g = 0$$

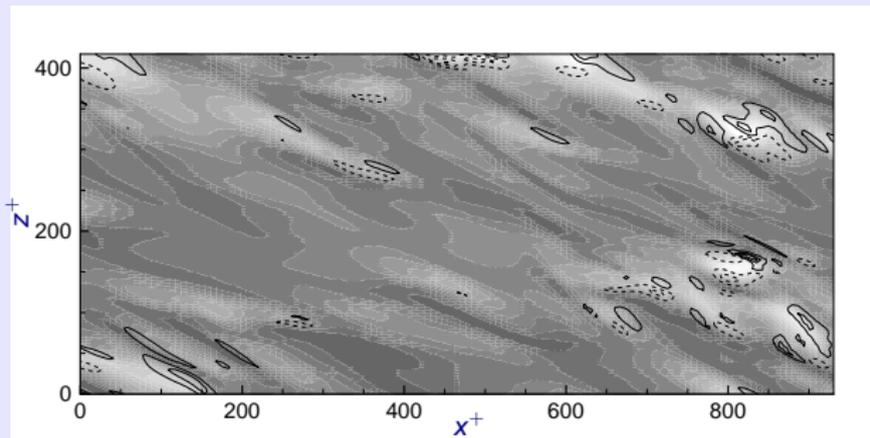
### TOTAL KINETIC ENERGY BALANCE

$$U_b^+ \tau_w^+ + \varepsilon_w^+ = \mathcal{D}_U^+ + \mathcal{D}_W^+ + \mathcal{D}_T^+$$

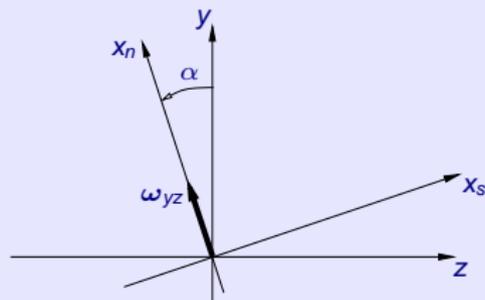
### TURBULENT DISSIPATION

$$\mathcal{D}_T^+ = \left[ \widehat{\omega_i \omega_i} \right]_g^+$$

- $\widehat{\omega}_z \widehat{\omega}_y \partial \widehat{W} / \partial y$  is key term leading to drag reduction
- $\widehat{\omega}_z \widehat{\omega}_y \partial \widehat{W} / \partial y \rightarrow \partial \widehat{W} / \partial y$  acts on  $\widehat{\omega}_z \widehat{\omega}_y$
- $\widehat{\omega}_z \widehat{\omega}_y \approx \frac{\partial u}{\partial y} \frac{\partial u}{\partial z}$ 
  - $\frac{\partial u}{\partial y} \rightarrow$  upward eruption of near-wall low-speed fluid
  - $\frac{\partial u}{\partial z} \rightarrow$  lateral flanks of the low-speed streaks



$\frac{\partial u}{\partial y} \frac{\partial u}{\partial z}$  located at the sides of high-speed streaks



## SIMPLIFIED TURBULENT ENSTROPY EQUATION

$$\frac{1}{2} \frac{\partial}{\partial t} (\omega_y^2 + \omega_z^2) = \omega_z \omega_y G - \left( \frac{\partial \omega_y}{\partial y} \right)^2 - \left( \frac{\partial \omega_z}{\partial y} \right)^2$$

Rotation of axis

$$\frac{1}{2} \frac{\partial \omega_n^2}{\partial t} = S_{nn} \omega_n^2 - \left( \frac{\partial \omega_n}{\partial y} \right)^2$$

Integration by Charpit's method

$$\omega_n = \omega_{n,0} \underbrace{e^{\sin \alpha \cos \alpha G t}}_{\text{stretching}} \underbrace{e^{-\beta^2 t} e^{-\beta y}}_{\text{dissipation}}, \beta = \frac{\partial \omega_n / \partial t}{\partial \omega_n / \partial y} \sim \frac{\lambda_y}{\lambda_t}$$